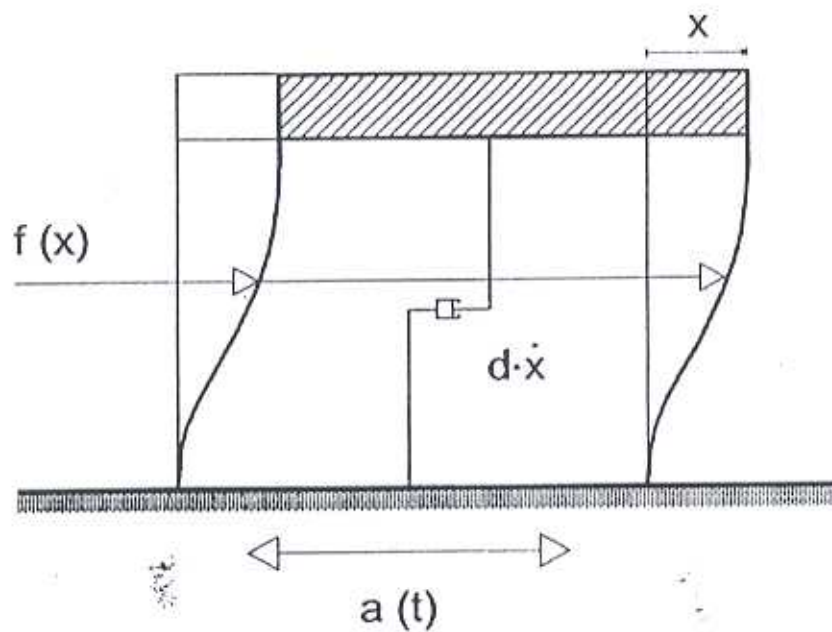


L'oscillatore semplice

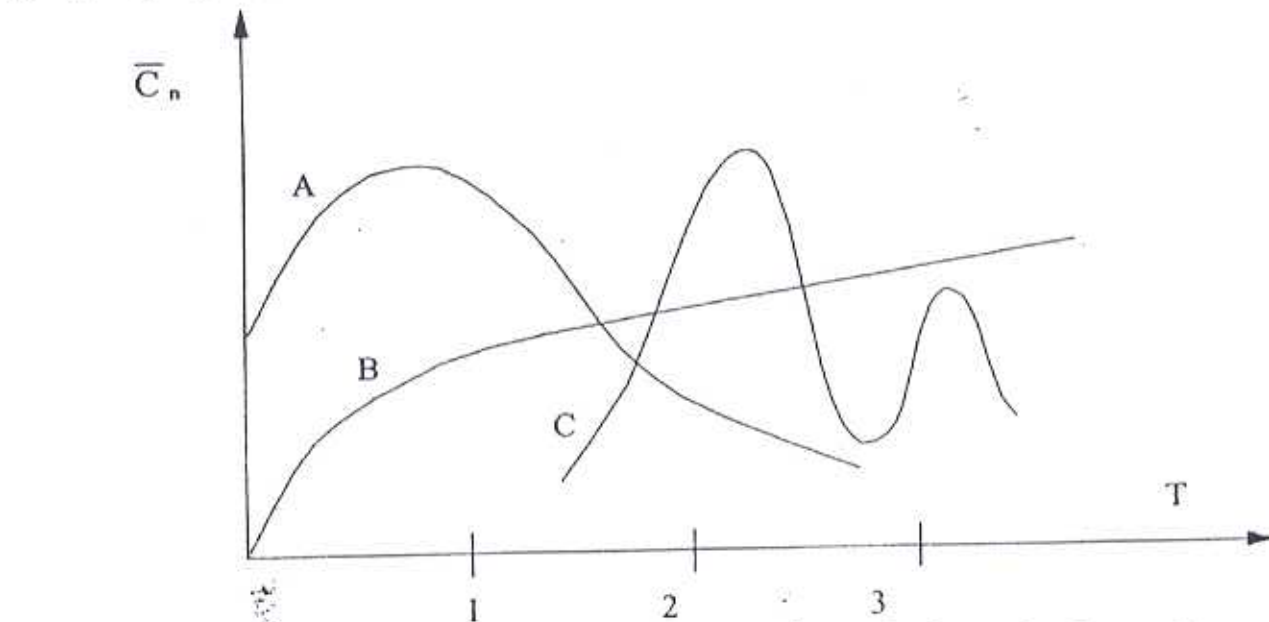


Periodo proprio

$$T = 2\pi \sqrt{\frac{m}{k}}$$

$$m\ddot{x} + d\dot{x} + f(x) = m \cdot a \cdot \sum_1^N \bar{C}_n \cdot \cos(\omega_n t - \varphi_n)$$

Spettri di Fourier di eventi sismici differenti



- A: su roccia, vicino (componenti fra 0.3 e 1 sec)
- B: terreno compatto, lontano (alte frequenze smorzate)
- C: terreni soffici, selezione delle armoniche

Intensità di $a(t)$

- Potenza media del sisma

$$W = \frac{1}{D} \int_0^D a^2(t) dt$$

$$W = \frac{1}{2} \sum_1^N C_n^2$$

- Intensità

$$I = \sqrt{W}$$

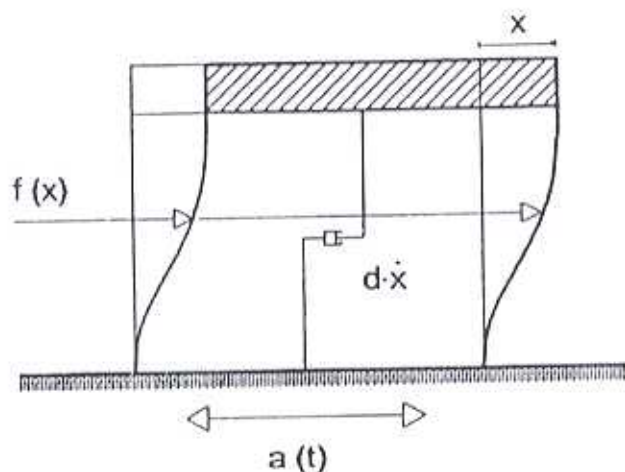
$$I = \sqrt{\frac{1}{2} \sum_1^N C_n^2}$$

- Il contributo di ogni armonica è $\frac{1}{2} C_n^2$

Lo spettro di risposta

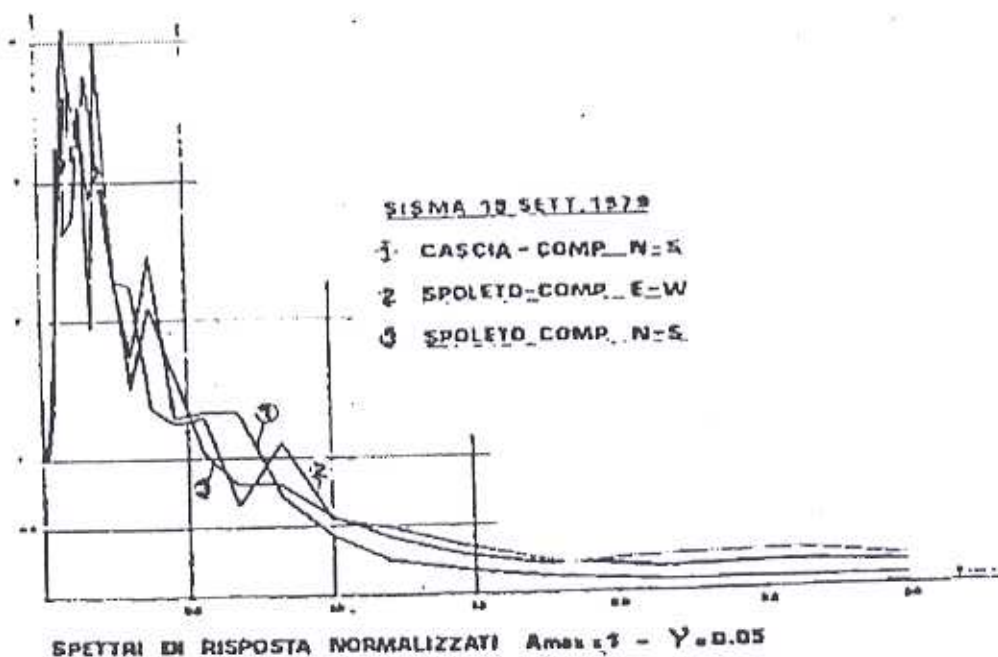
- E' l'insieme dei punti che rappresentano **la risposta massima di un oscillatore semplice sotto un dato accelerogramma, al variare del suo periodo T e per un dato smorzamento**
- La risposta può essere:
 - ▷ in accelerazione
 - ▷ in velocità
 - ▷ in spostamento

Lo spettro di risposta di un accelerogramma



$$F_{\max} = M \cdot S_a(\bar{T})$$

Della storia dinamica si conserva
solo il valore massimo



OSCILLATORE ELEMENTARE (Oscillazioni libere)

EQUAZIONE DEL MOTO : $M \frac{dx^2}{dt^2} + C \frac{dx}{dt} + Kx = 0$

Definizioni : M = massa, C = smorzamento viscoso, K = rigidezza

Posizioni : $\omega^2 = K/M$, $2\zeta\omega = C/M$

con ω = velocità angolare e ζ = coefficiente di smorzamento

EQUAZIONE DEL MOTO : $\frac{dx^2}{dt^2} + 2\zeta\omega \frac{dx}{dt} + \omega^2 x = 0$

$x = e^{pt} \Rightarrow p^2 + 2\zeta\omega p + \omega^2 = 0 \Rightarrow p_{1,2} = \omega(-\zeta \pm (\zeta^2 - 1)^{0,5})$

SOLUZIONE : $x = A_1 e^{p_1 t} + A_2 e^{p_2 t}$

A_1 ed A_2 costanti d'integrazione calcolate per : $x(t=0)$, $\frac{dx}{dt}(t=0)$

DISCUSSIONE

$\zeta \leq 1 \Rightarrow p_{1,2} = \omega(-\zeta \pm i(1-\zeta^2)^{0,5})$

$x = e^{-\zeta\omega t} (A_1 e^{i\sqrt{1-\zeta^2}\omega t} + A_2 e^{-i\sqrt{1-\zeta^2}\omega t})$

Ponendo:

$\omega_D = \omega \sqrt{1-\zeta^2}$,

$T = \text{Periodo proprio} = T = \frac{2\pi}{\omega}$, $T_D = \frac{2\pi}{\omega_D}$,

$v = \text{Frequenza} = v = \frac{1}{T}$ $v_D = \frac{1}{T_D}$

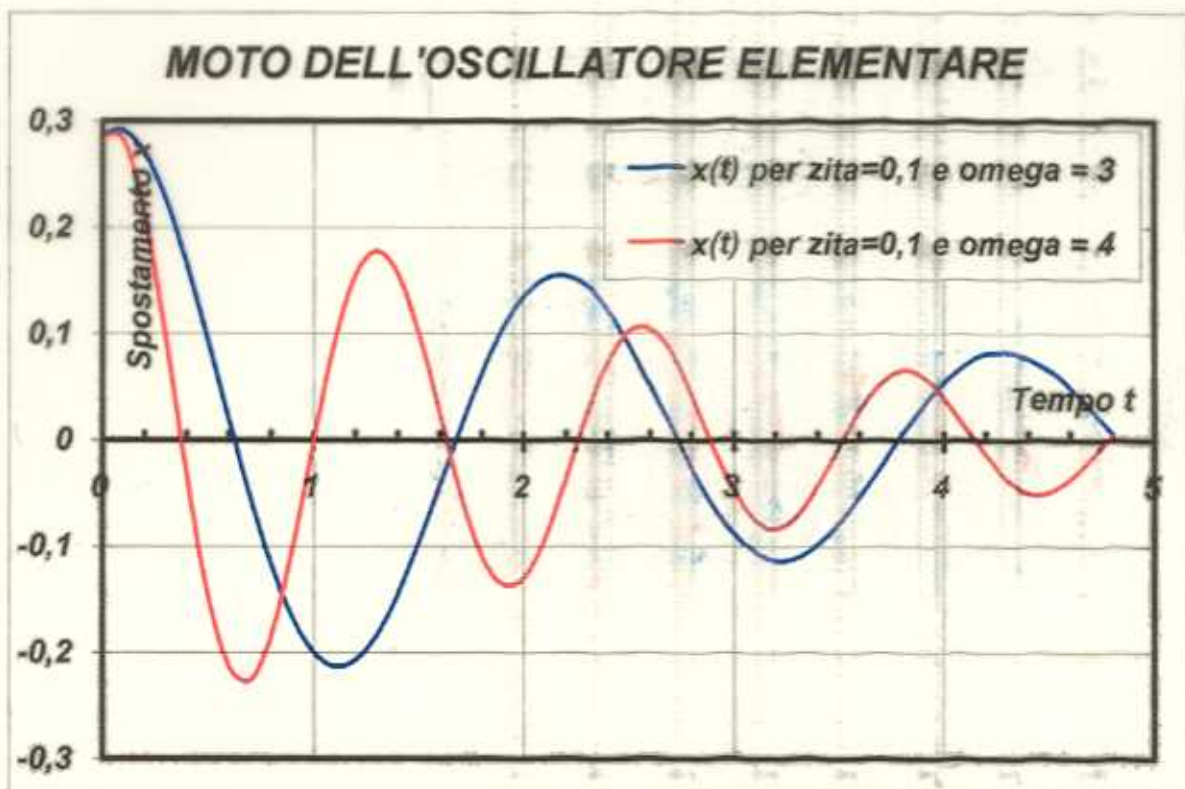
Le costanti di integrazione diventano:

$$a = \text{ampiezza} = \left[x(t=0)^2 + \left(\frac{\frac{dx}{dt}(t=0) + \zeta \omega x(t=0)}{\omega_D} \right)^2 \right]^{1/2}$$

$$\varphi = \text{angolo di fase} = \tan^{-1} \left[\frac{\omega}{\omega_D} \zeta + \frac{\frac{dx}{dt}(t=0)}{\omega_D x(t=0)} \right]$$

SOLUZIONE:

$$x(t) = a e^{-\zeta \omega t} \cos(\omega_D t - \varphi)$$



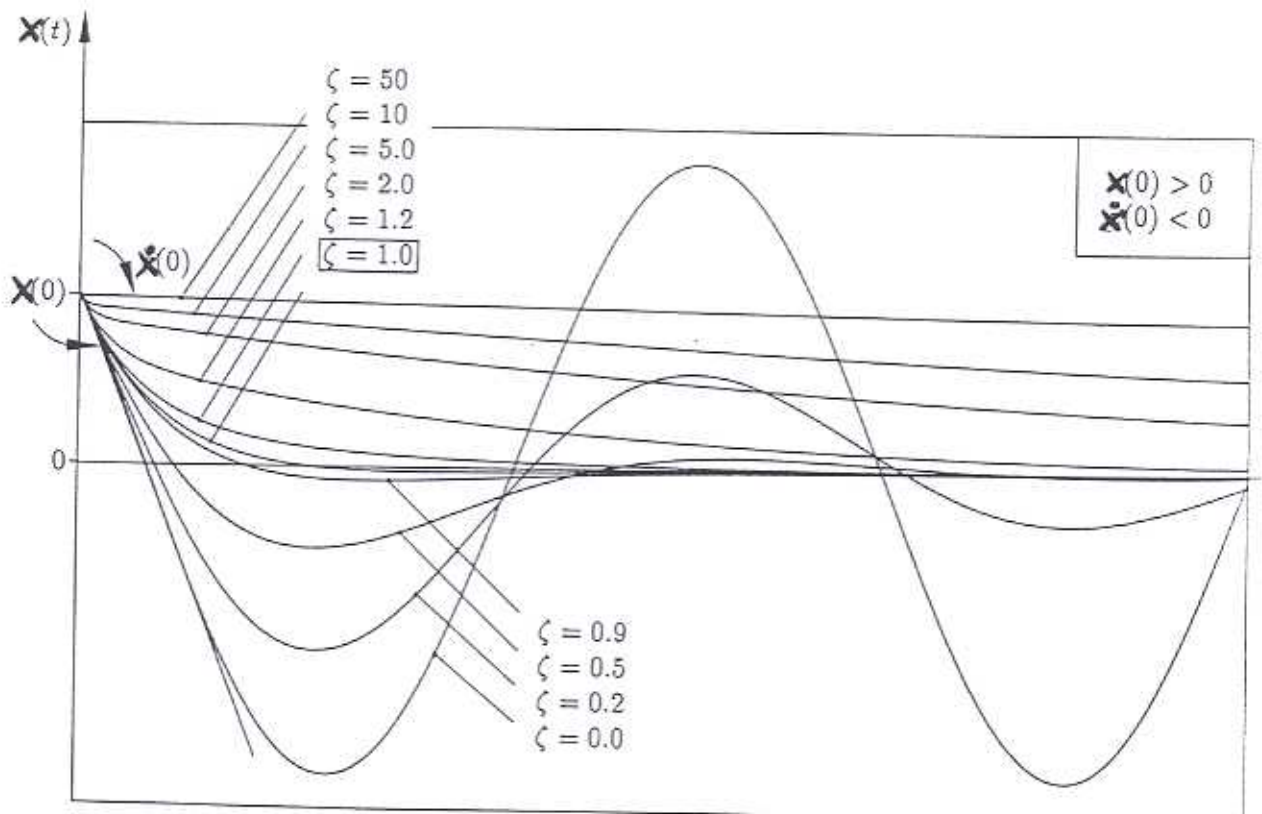
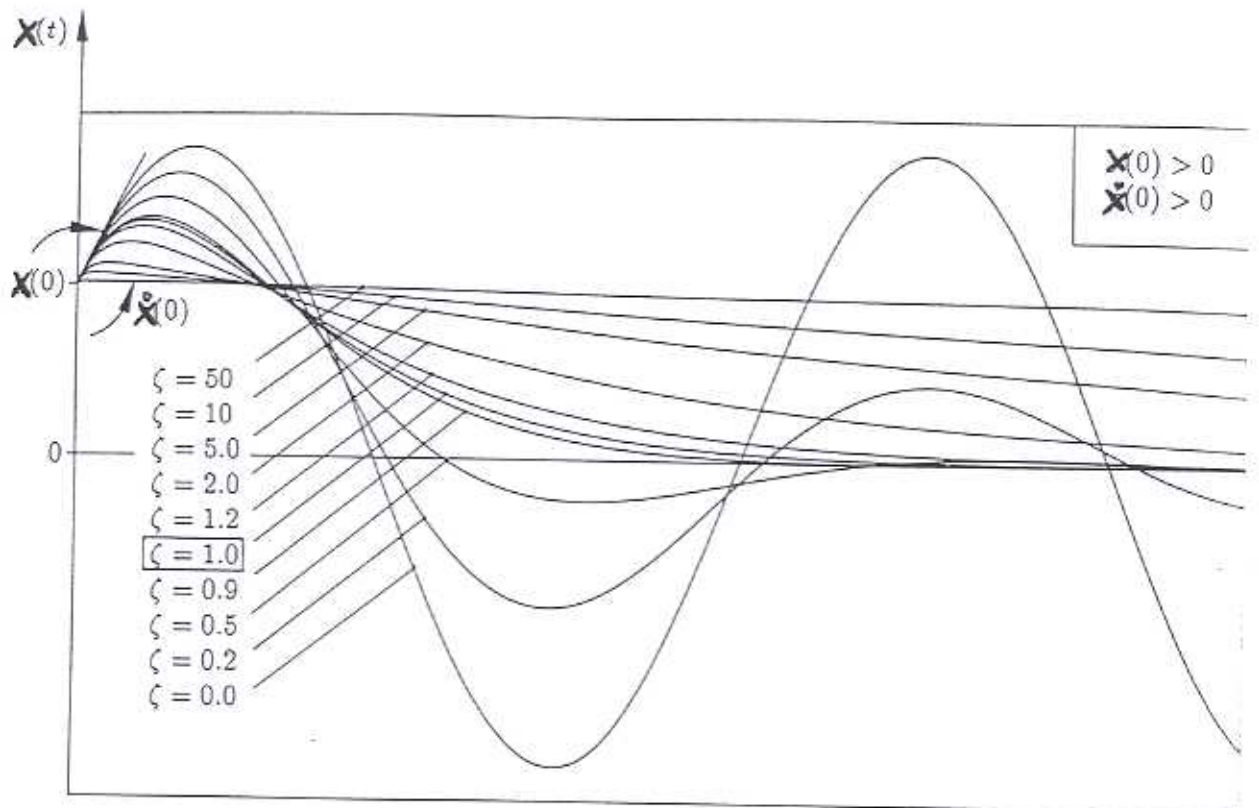


Fig. 1.3.2 Characteristic responses of an oscillator with viscous damping;
Part 1: sub-critical, critical, over-critical damping

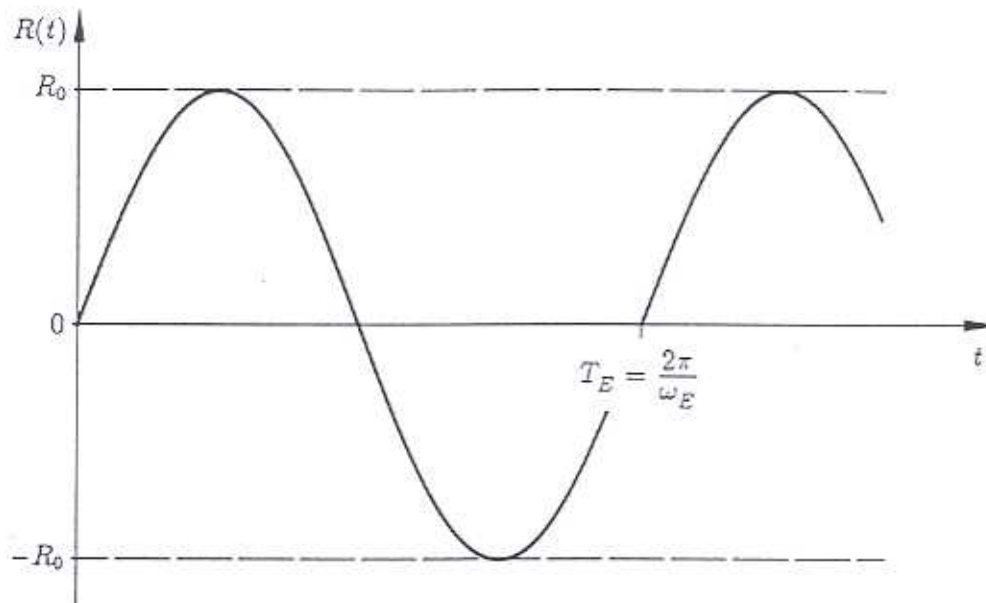


Fig. 1.4.1 Harmonic force excitation

Thus, for extremely slow oscillations, we would obtain a maximum deflection of $\pm X_0$. We now seek a particular solution X_p for

$$\ddot{x} + 2\zeta\omega\dot{x} + \omega^2x = \omega^2X_0 \sin \omega_E t \quad (1.4.5)$$

and assume that

$$X_p(t) = a_p \sin \omega_E t + b_p \cos \omega_E t \quad (1.4.6)$$

Substituting this expression into equ. (1.4.5) and comparing coefficients, we obtain upon introduction of the frequency ratio

$$\beta = \frac{\omega_E}{\omega} \quad (1.4.7)$$

two linear simultaneous equations for the constants a_p and b_p

$$\begin{bmatrix} 1 - \beta^2 & -2\zeta\beta \\ 2\zeta\beta & 1 - \beta^2 \end{bmatrix} \begin{bmatrix} a_p \\ b_p \end{bmatrix} = \begin{bmatrix} r_0 \\ 0 \end{bmatrix} \quad (1.4.8)$$

Their solution yields

$$\begin{aligned} a_p &= \frac{1 - \beta^2}{(2\zeta\beta)^2 + (1 - \beta^2)^2} r_0 \\ b_p &= \frac{-2\zeta\beta}{(2\zeta\beta)^2 + (1 - \beta^2)^2} r_0 \end{aligned} \quad (1.4.9)$$

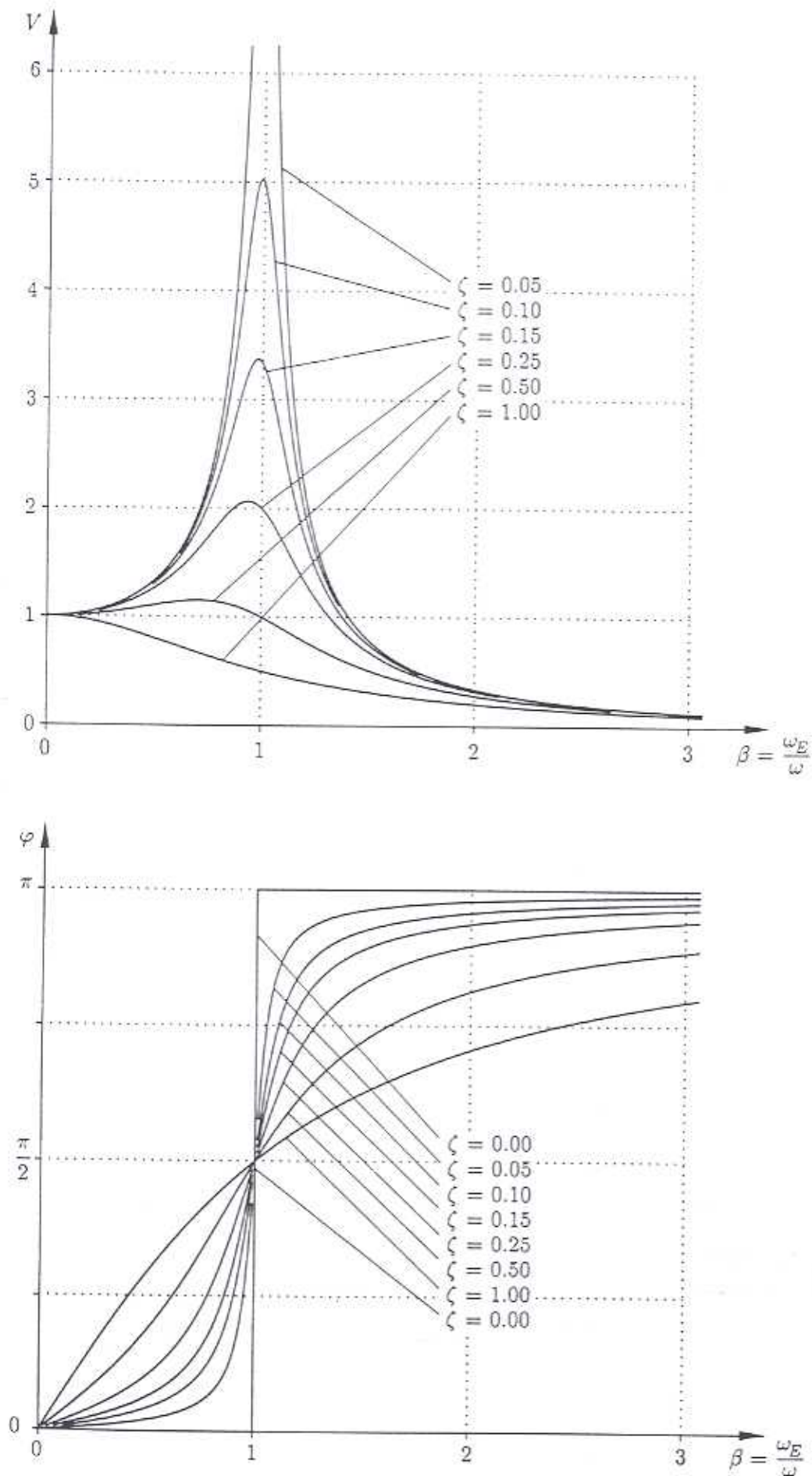


Fig. 1.4.2 Amplification factor and phase angle for the steady-state response of a harmonically excited system with viscous damping